Program



For use with Windows 95/98/ME/NT 4.0/2000/XP

Additional RSTAB-Module For Stress Analyses according AISC-ASD-9th Edition

User Manual

Version: June 2002

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1. Introduction

1.1. About ASD for Windows

ASD was developed according to the "plug & play" principle: you don't need a lot of previous knowledge to use the program. Everyone, experienced or not, should be able to get their job done with ASD. The practical design of the software was developed through the input of many customers and engineers. We thank those people, and encourage every ASD user to comment on the program.

ASD for Windows is integrated in the user interface of RSTAB 5 for Windows . Results of the dimensioning process with ASD are available in the RSTAB printout report. This enables the user to present their structural analysis documents in a uniform and attractive manner. ASD also allows the user to optimize sections and to export them into RSTAB for repetitive calculations. RSTAB's section library is fully supported, and all sections that are added there can be dimensioned. All sections generated with SHAPE and DICKQ – the section property tools of Ing.-Software Dlubal GmbH – are also supported. Furthermore, ASD has been programmed with 32 bit technology, which results in a extremely fast stress calculations. Finally, the use of the online help system is encouraged. It can be accessed with the [F1] key.

We hope RSTAB and ASD will contribute to the success of your work and company.

2. The ASD Development Team

The following people took part in developing ASD for Windows:

- Program Coordination: Dipl.-Ing. Georg Dlubal Dipl.-Ing. (FH) Walter Rustler Ing. Pavel Bartoš
- **Programming:** Petr Snopek Dr.-Ing. Jaroslav Lain
- **Program Supervision and Quality Control:** Dipl.-Ing. (FH) Walter Rustler Robert Gannon B.Eng. , M.Sc
- Manual and Help System: Robert Gannon B.Eng., M.Sc.

3. ASD Installation

3.1. System Requirements

To use RSTAB5 and ASD comfortably, we recommend the following minimum system requirements:

- Operating System Windows 95 / 98 / NT 4.0
- Processor with 200 MHz
- 32 Megabyte RAM
- CD-ROM- and 3,5" disk drive for installation
- 2 Gigabyte total hard disk capacity with about 50 Megabyte reserved for installation
- Graphic card with a resolution of 1024 x 768 pixels

With the exception of the operating system, no other product recommendations are made. Basically, RSTAB runs on all systems that fulfill the system requirements. Your computer does not need to have "Intel Inside", and it is also not necessary to have an expensive 3D graphic card. Because RSTAB and ASD are generally used for intensive calculations, the phrase, "more is better", holds true.

3.2. Installation Process

As a licensed ASD user, choose [Standard] at the point during the installation process to install all licensed programs – including ASD. Simultaneously, all other available additional modules will be installed as limited demo versions.

Installation Type		×
	Choose the Typ	e of Installation: Standard Al Licensed Programs and Demo Versions of not Licensed Programs Minimum Only licensed programs User Defined Choose the programs, that will be installed
		< <u>B</u> ack <u>N</u> ext> Cancel

Installation Type



If you lack free hard disk capacity, or it needs to be saved, choose [Minimum]. Only licensed full versions will then be installed. The rest of the installation process, including the authorization process, is the same as for RSTAB, explained in detail in the RSTAB manual.

3.3. Working with ASD

3.4. Starting ASD

The ASD module can be started from either the pulldown menu Additional Modules $\rightarrow Design \rightarrow ASD$, or through the tree item [Additional Modules] in the Position Navigator or Project Navigator.





 $/ \mathbb{N}$

Starting ASD with the Pulldown Menu Additional Modules, or the Navigator

Starting ASD independently is not possible because it is fully integrated into RSTAB. This means that for any design to be carried out in ASD, the respective position must first be opened in RSTAB.

ASD enables the use of several *ASD Cases* for evaluating the internal forces calculated by RSTAB. This means that in the individual *ASD Cases*, any elements, load systems, load system groups or load system combinations can be assembled for stress analysis and checked against one or several allowable stresses. For example, different materials can be examined this way. By comparing the total weight in the parts list, the most efficient construction can be found immediately.

3.5. Masks

The input of *ASD Cases*, as well as the output of the results on the screen, is carried out in masks. After starting ASD, the *ASD Navigator* appears to the left. It enables viewing and accessing all available input and output masks. At the top, a pull down list box with all available *ASD CASES* can be found. The list box can be opened by clicking on the [Arrow down] key, after which the appropriate *ASD CASE* can be selected. Under the title bar the three pulldown menus, *File, Edit* and *Help* are found. Those menus are explained in detail in chapter 3.6.



•

The access to all masks is controlled either by clicking on the respective item in the *ASD Navigator* or by flipping through the masks in succession. Use the [F2] and [F3] keys or the buttons [<<] and [>>] to flip between the masks. Access to the the libraries – if relevant - is possible with the [F7] key or by clicking the right mouse button.

With [Graphic], the graphic display of the results of the current ASD case is activated. More about graphical display and output of the results is found in chapter 4.

[OK] saves the input and results of ASD and closes ASD, while [Cancel] closes ASD without saving any editing. The button [Help] or the [F1] key starts the online help system.

3.6. Input Masks

In the input masks, all neccessary entries for the stress analysis and several parameter adjustments must be made.

3.7. Mask 1.1 General Data

After starting ASD, the window opens with mask 1.1 General Data.



Mask 1.1 General Data

The option *Elements to Design* is a default setting set on *All*. This can be deactivated by clicking on the check mark in the box to the left of *All*. Then the input line can be edited and the numbers of the elements to design can be entered manually. To make this selection graphically, use the [Pick] button.

The lists for *Existing Load Systems, LS Groups and LS Combinations* and *Selected for Design* are located in the middle section. To select one or several load systems, LS groups or LS combinations for the stress analysis, just double click on your choice from the lists. It will be moved to the *To Design* list. This can also be done by marking your selected load systems, LS groups and LS combinations and then using the button [>]. If the button [>>] is used, all existing load systems, LS groups or LS combinations are moved to the right. To remove single or all load systems, LS groups or LS combinations, use the buttons [<] or [<<] in the same way.

In addition, it is possible to add *Comments* to every ASD case. Before the [Calculation] is started, check [Details] for the correct settings of several parameters. Usually they are set on an appropriate default setting.



ASD, Details



Calculation

<u>P</u>ick

> >>

< <<



•

The result masks 2.1 to 2.7 can be activated under **Display Result Masks**. In mask 2.6, Governing Internal Forces for, you can also select for which stress type the governing internal forces are to be displayed. Use the [Arrow Down] key to change the selection in the listbox. In 2.7 Parts List, check whether you want the parts list Of All Elements or Only of Dimensioned Elements.

Under *Stresses to Calculate*, you can activate the desired stress type by clicking in the appropriate check boxes. [All] will activate all stress types.

Under *Units*, the required units for *Stresses, Single Weight* and *Total Weight* can be set from a list box with the use of the [Arrow Down] button. The most common values have been set to default, so usually no changes are necessary.

[Default] resets the default values to include all options. [OK] closes the window and adopts all changes. [Cancel] closes the window and discards all changes.

3.8. Mask 1.2 Materials and Sections

In this two-part mask, the allowable stresses are entered in the top section and these are used to check against the calculated stresses. For many materials, these allowable stresses are already stored in the materials library and are used automatically in ASD. At any time, however, you can change any of the values in mask *1.2.1 Material Parameters*, or make permanent changes in the library.

JI - Stress Analysis	1.2.1	'Aateria	Parameters					
Innut Data	-		A	B	С	D		E
General Data	Mat No	D	Material escription	Yield Point [kN/cm ²]	Tension Strength [kN/cm^2]	Modulus of I [kN/crr	Elasticity 1^2]	Code
Material Parameters and Sections	1	Steel A	36 🗾	24.82	39.99	19993.8		ASD, 9th edition
Design Parameters	2	Steel A572 Grade 50		34.47	44.81	1	9993.80	ASD, 9th Edition
	1226	action						Material Parameters Librar
	1.2.2	A		P			185 500	
	Sec No	Mat No		Section Description	Opti mize	Remark	1 2 300	
	1	1	IPE 500					
	2	1	IPE 360					Î I
	3	1	IPE 450					
	5	1	HE-A 160		<u>_</u>			
		1	HE-A 120		<u>_</u>			
	10	2	IPE 360		<u>H</u>			
	12	1	0E0.90-4					
	13	1	Circle 24					
	15	1	HE-A 200		- H			
	Comme	ents						3,=
					Section 9 Details	Section Library	Stress	: Points Display 🔲 Numbr

Mask 1.2 Materials and Sections

Material Parameters Library...

If the cursor is placed in the table of mask 1.2.1, the section library can be accessed by clicking the right mouse button or by using [F7]. The button [Material Parameters Library] can also be used. More detailed explanations about this library are found in chapter 3.6.2.

The lower section, **1.2.2** Sections, lists all sections to design including the respective *Material Number*. Use column C to allow the section to be *Optimized*. There are two ways to set the *Optimize* Option. Either click directly in the box in column C, or click on the [Arrow down] button to reveal a list and then select "Yes". Deactivating the *Optimize* Option is done the same way. During the calculation, the most suitable section of the relevant table is found should the optimization option be enabled. ASD uses the previously calculated internal forces of RSTAB (using the stiffness properties of the sections originally chosen in RSTAB) to find the required moment of inertia, and selects the section that lies just below a capacity utilisation of 100 %. In column D, *Remark*, you'll find the annotation 3) and in Comments at the bottom of mask 1.2.2 appears a comment: 3) The section will be optimized! The most optimal section of the table will be sought out!

If the annotation is 7), it means: 7) The section will not be designed because no elements use this section!

Annotation 1) means: 1) The section will not be designed because its section data is not defined! In this case, you must return to RSTAB and enter the necessary data. After optimization, you may need to recalculate the RSTAB results with the new optimized sections. The effect of the changed stiffnesses in the structure may have an effect on the internal forces, especially if a second-order (P-Delta) analysis has been carried out. The transferal of the optimized section to RSTAB is done via the pull-down menu, $Edit \rightarrow Export Optimized section to RSTAB$. Just recalculate the stresses in ASD again and the RSTAB results will be calculated at the same time. It may also be necessary to optimize them once more. With this repetitive action, you should be able to find the best and most efficient section.

[Section Properties] displays the relevant statical data of the current section, while [Section Library] provides access to the section library, which should already be familiar from RSTAB.

In the right hand window of mask 1.2.2, the section is displayed, together with its *Stress Points* and their *Numbering* blended in, if desired.

3.9. Mask 1.3 Design Parameters

In this mask, relevant design criteria pertaining primarily to stability conditions are displayed. Each of the tabulated values can be edited by the user.

\SD - [Portal Frame]											j.
File Edit Help											
ASD1 - Stress Analysis 🛛 💌	1.3 De	sign Paramel	ters								
⊡- Input Data		A	<u>B</u>	С	D	E	<u> </u>	G	H		J
- General Data	Elem No	Length	Effective-ler K2	Ngth factor	Coeff Cm2	Cm3	Coefficient Ch	Unbrace 1 h2 [m]	Length	Comp. Length	Anet/Agross
Material Parameters and	1	3.000	1.000	1.000	1.00	1.00	1.00	0.001	0.001	0.001	1.0000
····· Design Parameters	2	5.099	1.000	1.000	1.00	1.00	1.00	0.001	0.001	0.001	1.0000
	3	5.099	1.000	1.000	1.00	1.00	1.00	0.001	0.001	0.001	1.0000
	4	3.000	1.000	1.000	1.00	1.00	1.00	0.001	0.001	0.001	1.0000
Calculation Details	Setu	p		<< _>>>	_	Grap	hic		К	Cancel	Help

Mask 1.2 Materials and Sections

In column A the nominal element length is read in automatically from RSTAB. These lengths do not necessarily refer to the actual unbraced or effective length of the member to be designed, but rather to the distance between neighboring nodes possibly used to divide the member into smaller parts. It is therefore possible to enter in columns B and C the effective length factors K2 and K3 which refer to the strong and weak axes respectively. The unbraced lengths, required for flexural-torsional buckling, about the strong and weak axes can be entered in columns G and H respectively. These lengths are usually set to 1.00 as default.

The values for Cm2 and Cm3, modification factors relating to bracing against joint translation and member end rotational restraint, can be either manually entered or be alternatively selected from a dropdown listbox which opens after clicking on the arrow which appears when the the cell is activated.



Sections Library... The coefficient Cb in column F accounts for the variation in bending moment at the member ends. This value may be taken conservatively as 1 or can be calculated automatically from the program via the dropdown listbox available in this cell.

Lcomp can be entered in column I. This refers to the distance between points of conraflexure

The ratio of Agross to Anet can be entered in column J.

Should the user wish to change a certain number of the default values on any particular section(s), then it's advisable to click on [Set up]. Here, the user can edit the given values and apply them to the elements chosen via the pick function.

ASD Design Parameters		×
Elements to change		
Elements: 4,7,13,16,19,	20,54,55,58-74	Pick
Longeno	© K-values K2-2 0.80 K3-3	2.00
Set Effective Length	C Deview Lew - We K22×1 0000 m K22×1 0	0.000 m
	O Design Lengths K2-2 L 0.000 III K3-3 L 0	
Set Lb2 0.000 m	🗌 Set Lb3 🛛 0.000 m 🔲 Set Lcomp 🗍	0.000 m
Coefficients Cm, Cb		
Set Cm2-2 0.00	Set Cm3-3 0.00	3.00
Effective Area	Default Lengths and Coefficients	
Set Anet/Agross Ratio	0.82000 Set All Lengths as Element Length	
	Coefficients taken from Beam End M	oments
Apply	OK Cancel	Help

<u>Calculation</u>

3.10. Calculation

After starting the [Calculation], ASD searches for the necessary RSTAB results. If none are available, RSTAB automatically calculates the internal forces. Note that RSTAB *Calculation Parameters* are used during this action. If activated, the optimization of the sections is carried out and the stresses of all sections are determined. In the case of tapered elements, only the beginning and ending sections can be optimized. The moments of inertia as well as the stresses are then calculated for intermediate points. This method is an exact calculation of all necessary statical values. It is unnecessary to enter intermediate nodes and sections.

ASD uses the following equations in determining the relevant stresses.

Normal stress σ (positive = Tension, negative = Compression) caused by axial force N and bending moment M_y and M_z:

Tension Members

$$f_{t\,Gross} = \frac{N}{A_{Gross}}$$

$$f_{t\,Net} = \frac{N}{A_{Net}}$$

AGrossGross cross-sectional AreaANetNet cross-sectional Area

and

Compression Members

$$f_c = \frac{N}{A_{Gross}}$$

Beams and other Flexural Members

$$f_{b2} = \frac{M_2}{S_{2i}}$$
 Bending about the strong axis 2-2

$$f_{b3} = \frac{M_3}{S_{3i}}$$
 Bending about the weak axis 3-3

 $S_{2,i}$, $S_{3,i}$ Elastic Section Moduli in accordance with:

$$S_{2,i} = \frac{I_2}{e_{3,i}}$$
$$S_{3,i} = \frac{I_3}{e_{2,i}}$$
$$I_{2}I_3$$

 $e_{2,i}, e_{3,i}$

Moments of Inertia refering to the axes 2,

Distance (Eccentricity) from the Center of

to the relevant stress point in direc-

3,

3

Shear Stress τ due to Shear Forces V_2 and V_3 and Torsional Moments T: $\tau=\tau_V+\tau_T$

 τ_V Shear Stress component according to the following equation:

$$\tau_{V,i} = \frac{V_2 \cdot Q_{3,i}}{I_3 \cdot t_i} + \frac{V_3 \cdot Q_{2,i}}{I_2 \cdot t_i}$$

$$t_i \qquad \text{Thickness of the section at Stress Point i}$$

$$Q_{2,i} \qquad Q_{3,i} \qquad \text{Statical Moments at Stress Point i referring to axes 2 and}$$

 $\tau_{\rm T}$ Contribution of Torsion Moment with T

$$\tau_{T,i} = \frac{T}{J} \cdot t_i \text{ for thin walled open sections (St Venant) and}$$

$$\tau_{T,i} = \frac{T}{2 \cdot A_{m,i} \cdot t_i} \text{ for closed tubular sections (Bredt).}$$

J Torsional Constant
A_{m,I} mean enclosed area for Stress Point i.

The formulae for determining the allowable stresses are listed in Appendix A.

3.11. Result Masks

All results are displayed in result masks 2.1 to 2.5, inclusively. The section under consideration is displayed in a graphic to the right. The stresses displayed in this graphic change to correspond with the stress item selected in the tabular section to the left. The graphic can be shown with or without numbering or values and can be enlarged using the magnifying glass.



Results mask

By clicking on **[Details]**, the user gains direct access to tabulated section properties and section details in graphical format. In the graphic it's possible to display the designated stress points, the so called b/t zones and their respective numbering. By clicking on [Values] the user obtains values of the statical moments or details with respect to the b/t –zones.

Details of IPE 550									×
Description	Name	Size	Unit			IPE 550			
Depth	d	550.00	mm						
Width	Ьf	210.00	mm	11					
Web thickness	tw	11.10	mm				210.0		
Flange thickness	Ū	17.20	mm				1 1 2 3 4 2 5		
Fillet radius	r	24.00	mm			t t	Color Color		
Cross-section area	A	134.00	cm^2				2 1		
Shear area	A_2	60.47	cm^2				-		
Shear area	A_3	57.08	cm^2						
Web area	A_Web	57.20	cm^2			_			
Moment of inertia	ي ا	67120.00	cm^4			8	×		
Moment of inertia	I_z	2670.00	cm^4			55	13		2,9
Governing radius of gyration	U U	22.30	cm						
Governing radius of gyration	r_z	4.45	cm				111.1		
Polar radius of gyration	ĹΡ	22.74	cm						
Trägheitsradius des Gurtquerschnitt	r_zg	5.15	cm						
Nominal weight	Wt	105.19	kg/m			⊢ ∔	C 7		
Surface	A_Surf	1.88	m^2/m				6 37 694 10		
Torsional constant	J	124.00	cm^4				3,z		
Warping constant	C_w	1.884E+06	cm^6						
Elastic section modulus	S_y	2440.00	cm^3						
Elastic section modulus	S_z	254.00	cm^3						
Warping constant moment	W_Om_ma	6735.66	cm^4			Stress Points		E	AN
Statical moment	ر ۹	1390.00	cm^3		ľ		Values		
Statical moment	Q_z	94.82	cm^3		1	 Numbering 	values		
Normalized warping constant	Omega_ma	279.72	cm^2			— a w			
Warping statical moment	A_Om_ma>	2525.87	cm^4		1	I✓ (b/t)-zones			
Plastic section modulus	Wpl_2	2780.00	cm^3			Numbering	Values		
Plastic section modulus	Wpl_3	400.54	cm^3						
Plastic shape factor	Alpha_pl_2	1.14					Connel	1	11-16 L
Plastic shape factor	Alpha_pl_3	1.58					Lancel		пер

Section Details

The section, together with its section properties, details and computed stresses, can be printed either directly to the printer, to clipboard or alternatively to a printout report. By clicking on the printer icon a new dialog appears with a list of all the printable items, from which the user can make a suitable selection



Printout setup

3.12. Mask 2.1 Max Stresses by Section

SD1 - Stress Analysis 🛛 💌	2.1 Max Stresses	by Se	ections							
Input Data General Data Material Parameters and Design Parameters Results Max Stresses by Section Max Stresses in Element Max Stresses in Locati	A Stress Type Section No 1 - IPE Tension Compression Bending Stress 2- Bending Stress 3- Charse Stress 3-	B Elem No 550 1 2 2 2 1	C × Location [m] 0.000 0.000 6.000 0.000	D S Point 1 1 1 1	E LS No LG1 LC1 -N LG1 LG1	F Stress [k existing 0.00 -1.02 -20.72 0.00	G N/cm^2] allowable 14.89 5.70 9.21 18.61	H Stress Ratio 0.00 0.18 2.25 0.00		IPE 550 Bending Stress 2-2 Element No 2, x = 6.000 [m]
— Governing Internal Force — Parts List	Shear Stress 2 Shear Stress 3 Torsion Stress Tension and Bend Compression and Section No 2 - IPE Tension Compression	1 2 1 1 2 450 4 5	0.000 0.000 0.000 6.000 0.000 10.130	1 13 1 1 1 1 1 1	LG1 LG1 LG1 LG1 LG1 LG1 LG1·N	0.00 1.71 0.00 0.00 0.00 -0.91	9.93 9.93 9.93 14.89 1.70	0.00 0.17 0.00 0.00 2.43 0.00 0.53		-20722000000072 207220000072 20722000072072 6709 10
	Bending Stress 2- Bending Stress 3- Shear Stress 2 Shear Stress 3 Torsion Stress	5 4 5 4	10.130 0.000 0.000 10.130 0.000	6 1 1 13 1	LG1 LG1 LC1 -N LG1	-17.04 0.00 0.00 -2.64 0.00	5.03 18.61 9.93 9.93 9.93	3.39 0.00 0.00 0.27 0.00	-	Max = 20.72 [kN/cm^2] (10) Min = -20.72 [kN/cm^2] (5) Cb = 1.59
	Equation for allowat	ole stre	ss F1-6, be	am mei	nber with CO	MPACT ele	ements			Details Section Stress Numbering Values

Mask 2.1 Max Stresses by Section

Mask *2.1 Stresses by Section*, contains the results of the max stresses for every element, Load System, Load Group and Load Combination selected in mask 1.1 under [Details].. The results are sorted by section.

For each *Stress Type* the *Element No, x Location, S Point No, LS No,Stress exist(ing), Stress allow(able)* and *Stress Ratio* are displayed. The so called stress ratio is the stress utilization factor and is the quotient relating existing to allowable stresses.

The stress ratio of the section in the last column is calculated according to the setting in mask 1.1 under [Details] of *Governing Internal Forces for*. If the ratio is greater

than 1, the number is printed in bold font for quick reference. The max. stress of the various individual stress components may be located at different points. Therefore, please note that you cannot necessarily add the results of the single stress components in the lines unless they all occur at the same stress point.

3.12.1. Mask 2.2 Max Stresses by Set of Elements

Stresses can be sorted according to the Sets of Elements defined in RSTAB mask 2.2. In addition, the relevant Element No. of the *Set of Elements* is listed in Column B.

5D1 - Stress Analysis 📃 💌	2.2 Max Stresses in	Sets	of Elemen	ts					
Innut Data	A	В	С	D	E	F	G	Н	IPE 450
- General Data	Stress	Elem	x Location	S	LS	Stress [k	(N/cm^2]	Stress	Compression
Material Parameters and	Туре	No	[m]	Point	No	existing	allowable	Ratio	Lienen no s, x = 10.150 [ii]
Design Parameters	Set of Element No 1: ,	Eleme	nts 3-6						
Besults	Tension	3	0.000	1	LG1	0.00	14.89	0.00	1 2 3 4 5
May Stresses bu Section	Compression	5	10.130	1	LC1 -N	-0.91	1.70	0.53	
May Stresses in Sets of	Bending Stress 2-2	5	10.130	6	LG1	-17.04	5.03	3.39	
May Stresses in Element	Bending Stress 3-3	3	0.000	1	LG1	0.00	18.61	0.00	
May Stresses in v Locati	Shear Stress 2	3	0.000	1	LG1	0.00	9.52	0.00	-0.91-0-091391-0.91
Strasses in All Strass Po	Shear Stress 3	3	2.410	13	LC1 -N	2.56	9.52	0.27	-0 87/h s
- Gouerning Internal Fores	Torsion Stress	3	0.000	1	LG1	0.00	9.93	0.00	
- Rate Liet	Tension and Bending	6	2.410	1	LC1 -M-2			1.41	
	Compression and Ben	5	10.130	6	LC1 -M-2			3.98	
									-0.31-0.8831-0.91
									Max = -0.91 [kN/cm*2] (13)
F	Comments — Equation for allowable : exceeded value 200 (c	stress auses	E2-2, beam failure), sler	memb nderne	er with COMP. ss ratio K*1/r e	ACT eleme exceeded :	ents, Ib/rt ra 200 (compr	atio ession)	Details

Mask 2.2 Max Stresses by Set of Elements

3.12.2. Mask 2.3 Max Stresses by Elements

In mask 2.3 Max Stresses in Elements, the maximum stresses of each element are listed.

A Stress	В	C I	D	1 6	1		
Stress			U	L E	F	G	▲ IPE 550
_	x Location	S	LS	Stress [k	<n cm^2]<="" td=""><td>Stress</td><td>Element No 1, x = 6 000 [m]</td></n>	Stress	Element No 1, x = 6 000 [m]
lype	[m]	Point	No	exising	allowable	Hatio	
Element No 1: Section No	1 - IPE 55	0	1.01	0.00	14.00	0.00	
Tension	0.000	1	LGI	0.00	14.89	0.00	-
Compression	0.000	1	LC1-N	-1.02	5.70	0.18	1.271.22271.27
Bending Stress 2-2	6.000	6	LUTIN	-19.80	9.42	2.10	
Bending Stress 3-3	0.000	1	LG1	0.00	18.61	0.00	1/2/3/1/3
Shear Stress 2	0.000	1	LG1	0.00	9.93	0.00	
Shear Stress 3	0.000	13	LC1 -N	-1.52	9.93	0.15	- 1
Torsion Stress	0.000	1	LG1	0.00	9.93	0.00	- I I I I I I I I I I I I I I I I I I I
Tension and Bending	0.000	1	LG1			0.00	0 0 7
Compression and Bendin	6.000	6	LC1 -N			2.28	2 4.97
Element No 2: Section No	1 · IPE 55	0					20872228228
Tension	0.000	1	LG1	0.00	14.89	0.00	
Compression	0.000	1	LC1 -N	-1.02	5.70	0.18	
Bending Stress 2-2	6.000	1	LG1	-20.72	9.21	2.25	
Bending Stress 3-3	0.000	1	LG1	0.00	18.61	0.00	6 7 89 10
Shear Stress 2	0.000	1	LG1	0.00	9.93	0.00	Max = 2.28 [1] (10)
Shear Stress 3	0.000	13	LG1	1.71	9.93	0.17	Cb = 1.94
Torsion Stress	0.000	1	LG1	0.00	9.93	0.00	Cm2 = 1.00, Cm3 = 1.00
Comments							
Equation for allowable stre	ss H1-3, be	am me	mber with CON	IPACT ele	ements		
							Section Stre
	Lement No1: Section No Tennion Compression Bending Stress 2: Bending Stress 2: Shear Stress 2 Shear Stress 3 Torsion Stress Tension and Bending Compression Tension Compression Bending Stress 3: Shear Stress 2: Bending Stress 3: Shear Stress 2: Shear Stress 3: Shear Stress 4: Shear Stre	Element No 1: Section No 1: IPE 55 Tennion 0.000 Compression 0.000 Bending Stress 2.2 6.000 Bending Stress 3.3 0.000 Shear Stress 3 0.000 Tension and Bending 0.000 Compression and Bending 6.000 Compression and Bending 6.000 Compression 0.000 Compression 0.000 Compression 0.000 Compression 0.000 Shear Stress 2.2 6.000 Bending Stress 3.3 0.000 Shear Stress 2.4 0.000 Shear Stress 3.3 0.000 Contracts 0.000 Shear Stress 3.4 0.000 Contracts 0.000	Element No.1: Section No.1 IPE: SSU Tennion 0.000 1 Compression 0.000 1 Bending Stress 2.2 6.000 6 Bending Stress 3.3 0.000 1 Shear Stress 3.4 0.000 1 Tension Stress 0.000 1 Tension and Bending 6.000 1 Compression and Bending 6.000 1 Compression 0.000 1 Compression 0.000 1 Pending Stress 3.2 6.000 1 Shear Stress 2.2 6.000 1 Pending Stress 3.3 0.000 1 Shear Stress 2.4 0.000 1 Shear Stress 3.4 0.000 1 Shear Stress 3.4 0.000 1 Compression 0.000 1 Equation for allowable stress H1-3, beam me Equation for allowable stress H1-3, beam me	Element No 1: Section No 1: IPE 550 Tension 0.000 1 LG1 Compersion 0.000 1 LC1 Al Bending Stress 2-2 6.000 6 LC1 Al Bending Stress 3-3 0.000 1 LG1 Shear Stress 3 0.000 1 LG1 Tension and Bending 0.000 1 LG1 Compersion 0.000 1 LG1 Compersion 0.000 1 LG1 Pending Stress 2-2 6.000 1 LG1 Shear Stress 2 0.000 1 LG1 Shear Stress 3 0.000 1 LG1 Shear Stress 2 0.000 1 LG1 Shear Stress 3 0.000 1 LG1 Commerts 0.000 1 LG1	Element No.1: Section No.1 IPE 550 Tennion 0.000 1 LG1 0.000 Compression 0.000 1 LG1 0.000 Bending Stess 2.2 6.000 6 LC1+N -1.920 Bending Stess 3.3 0.000 1 LG1 0.000 Shear Stress 3. 0.000 1 LG1 0.000 Shear Stress 3. 0.000 1 LG1 0.000 Tension and Bending 0.000 1 LG1 0.000 Compression and Bending 0.000 1 LG1 0.000 Compression 0.000 1 LG1 0.000 Shear Stress 2 0.000 1 LG1 0.000 Shear Stress 3 0.000 1 LG1 0.000 Shear Stress 3	Element No.1: Section No.1: -IPE 550 Tennion 0.000 1 LG1 0.00 14.89 Compression 0.000 1 LC1-N 1.98 9.42 Bending Stress 2.2 6.000 6 LC1-N 1.98.9 9.42 Bending Stress 2.2 0.000 1 LG1 0.00 18.61 Shear Stress 3 0.000 1 LG1 0.00 18.61 Shear Stress 3 0.000 13 LC1-N 1.52 9.33 Tension and Bending 0.000 1 LG1 0.00 9.33 Tension and Bending 0.000 1 LG1 0.00 9.33 Compression 0.000 1 LG1 0.00 9.33 Compression 0.000 1 LG1 0.00 1.82 Compression 0.000 1 LG1 0.00 1.489 Compression 0.000 1 LG1 0.00 1.85 Shear Stress 3 0.000	Element No.1: Section No.1: Infl: 550 Tennion 0.000 1 LG1 0.00 14.89 0.00 Compression 0.000 1 LC1 N 1.02 570 0.18 Bending Stress 2.2 6.000 6 LC1 N 1.98 9.42 2.10 Bending Stress 3.3 0.000 1 LG1 0.00 1861 0.00 Shear Stress 3 0.000 1 LG1 0.00 9.33 0.00 Tension and Bending 0.000 1 LG1 0.00 9.33 0.00 Compression and Bending 6.000 1 LG1 0.00 9.33 0.00 Compression and Bending 0.000 1 LG1 0.00 9.33 0.00 Compression 0.000 1 LG1 0.00 1.88 0.00 Compression 0.000 1 LG1 0.00 1.88 0.00 Shear Stress 3 0.000 1 LG1 0

Mask 2.3 Max Stresses by Element

3.12.3. Mask 2.4 Max Stresses by x Location

Mask 2.4 Max Stresses by x Location lists all stresses in the RSTAB defined division points. These points may not necessarily correspond to the locations of the maximum internal forces. Therefore, the locations of the max existing values are listed as well.

"EEL1 - Stress Analysis 🛛 💌	2.4 Max Stresses	by x Loca	tion							
Input Data	A	В	С	D	E	F	G	•	IPE 180	
- General Data	Stress	x Location	S	LS	Stress [k	N/cm^2]	Stress		Sigma N+M-2 Element No 1 x = 3 f	60 ml
Materials and Sections	Туре	[m]	Point	No	exising	allowable	Ratio		Lichen no 1, x - o.	we ful
Paculto	Sigma Total	3.200	1	LC2 +M-2	-180.05	21.82	8.25			
May Stresses by Section	Tau due to V-2	3.200	1	LG1	0.00	12.60	0.00		4 9 9	
Max Stresses by Section	Tau due to V-3	3.200	13	LC2 +V-3	1.62	12.60	0.13		1 1111	ter t
Max Stiesses by Set of t	Tau due to V	3.200	13	LC2 +V-3	1.62	12.60	0.13		/////	777
Max Stresses by Elenier	Tau due to M-T St	3.200	1	LG1	0.00	12.60	0.00			11
Max Stresses by x Locatio	au due to M-T Br	3.200	1	LG1	0.00	12.60	0.00			Ζ.
- Governing Internal Forse	Tau due to M·T	3.200	1	LG1	0.00	12.60	0.00	1	-180.1 -146.6 2	10.1
- Parts List	Tau Total	3.200	13	LC2 +V-3	1.62	12.60	0.13	1	180	3446.0 18
	Sigma eg	3.200	3	LC2 +M-2	180.06	21.82	8.25	1	1 7	1000
	Element No 1: Sect	ion No. 1 - I	PE 180	x = 3.66m				1		7/////
	Sigma due to N	3.660	1	LG1	0.00	21.82	0.00	1		
	Sigma due to M-2	3.660	1	LG2	-180.05	21.82	8.25	1		411
	Sigma due to M-3	3.660	1	LG1	0.00	21.82	0.00	1	l ° (13 10
	Sigma N+M-2	3.660	1	LG2	-180.05	21.82	8.25	1		
	Sigma N+M-3	3 660	1	1.61	0.00	21.82	0.00	1		
	Sigma due to M	3 660	1	1.62	-180.05	21.82	8 25	1		
	Sigma Tensile	3 660	6	162	180.05	21.82	8 25	1	May - 100.05 R.A.	1/om^21(6)
	Sigma Compressiv	3,660	1	162	-180.05	21.82	8 25	1	Min = -180.05 [kN	l/cm^2] (1)
	Sigma Delta	3,660	1		180.05	21.02	0.20			and on t
	Sigma Total	3,660	1	1.62	-180.05	21.82	8 25	1	Details	य छ
	Tau due to V-2	3,660	1	LG1	0.00	12.60	0.00	1	C Section	Stresse
	Tau due to V-3	3,660	13	162	.1.23	12.00	0.00	-	Numberine	Values
•	r au uue tu v-3	3.000	10	Laz	-1.25	12.00	0.10	Ľ	I IV INumbering	I values

Mask 2.4 Max Stresses by x Location

3.12.4. Mask 2.5 Stresses at every Stress Point

The most detailed mask is mask 2.5 Stresses at every Stress Point. Here you can view all stresses for all defined stress points at all x-locations. The mask will only appear if it is enabled in mask 1.1 Details, in ASD. This is a large amount of data that is only necessary for specific examination. That's why this mask is disabled by default. The absolute max/min stresses, however, are always determined in ASD even if this mask is not selected for display.

ASD1 - Stress Analysis 🛛 💌	2.5 Stresses in All Stre	ss Points								
Input Data General Data General Data Material Parameters and Design Parameters Max Stresses by Section Max Stresses in Sets of Max Stresses in Sets of Max Stresses in Sets of	A Stress Type Element No 1, x = 0.00, S Tension Compression Bending Stress 2-2 Bending Stress 3-3	B x Location [m] Point No 1 0.000 0.000 0.000 0.000	C S Point (y=-10 1 1 1 1	D LS No LG1 LG1 -N LG1 LG1 LG1	E Stress [k exising 0.00 -1.02 0.00 0.00	F N/cm^2] allowable 14.89 5.70 14.89 18.61	G Stress Ratio 0.00 0.18 0.00 0.00		IPE 550 Tension Element No 1, x = 0.000 [m]	
Max Stresses in x Locat Stresses in All Stress Po Governing Internal Force Parts List	Shear Stress 2 Shear Stress 3 Torsion Stress Tension and Bending Compression and Bendin Element No 1, x = 0.00, S	0.000 0.000 0.000 0.000 0.000 Point No 2	1 1 1 1 (y=-2.5	LG1 LG1 LG1 LG1 LC1 -N 35, z=-27.50 ct	0.00 0.00 0.00 m)	9.93 9.93 9.93	0.00 0.00 0.00 0.00 0.18		1 3 — 1	
	Tension Compression Bending Stress 2-2 Bending Stress 3-3 Shear Stress 2 Shear Stress 3 Torsion Stress	0.000 0.000 0.000 0.000 0.000 0.000 0.000	2 2 2 2 2 2 2 2 2 2 2	LG1 LG1-N LG1 LG1 LG1 LG1-N LG1	0.00 -1.02 0.00 0.00 0.00 -0.24 0.00	14.89 5.70 14.89 18.61 9.93 9.93 9.93	0.00 0.18 0.00 0.00 0.00 0.02 0.02	•	Max = 0.00 [kN/cm^2](13) Min = 0.00 [kN/cm^2](13)	0
<►	Comments Equation for allowable stre	iss D1, on g	ross ar	ea, beam men	ber with C	OMPACT	element	s	Details (Section Vumbering	B Stress Values

Mask 2.5 Stresses at every Stress Point

3.12.5. Mask 2.6 Governing Internal Forces

This mask lists the governing internal forces for each element. These are the internal forces and moments that result in the largest stress for the element. In mask *1.1 General Data,* you can set to which stress type that these internal forces should be related by clicking on [Details].

ASD - [Demo-4] File Edit Help									3
ASD1 - Stress Analysis 💌	2.6 Gov	erning Internal	Forces						
E-Input Data	A	В	С	D	E	F	G	Н	
- General Data	Elem	x Location	LS		Forces [kN]			foments [kNm]	
 Material Parameters and 	No	[m]	No	N	V-2	V-3	M-I	M-2	M-3
Design Parameters	1	6.000	101	-127.92	0.00	-79.80	0.00	-483.19	0.00
- Results	2	6.000	LG1	-123.91	0.00	77.92	0.00	505.65	0.00
Max Stresses by Section	3	0.000	LG3	-94.62	0.00	119.36	0.00	-483.19	0.00
- Max Stresses in Sets of	4	9.120	1.01	0.00	0.00	0.00	0.00	0.00	0.00
— Max Stresses in Element	- U	2.410	1.61	-00.03	0.00	-30.30	0.00	-200.47	0.00
Parts List									
			<<	>>	Graphic		OK	Cancel	Help

Mask 2.6 Governing Internal Forces

3.12.6. Mask 2.7 Parts List

In this last mask, 2.7 *Parts List*, you get a summary of all sections used in the structure. When Sets of Elements are available, this mask is split into two windows. One window displays 2.7.1. *Parts List by Elements* and the other pane shows 2.7.2 *Parts List by Sets of Elements*.

ile Edit Help								
SD1 - Stress Analysis 📃 💌	2.7.1 Pa	rts List l	oy Elements					
Input Data	A	В	C	D	E	F	G	Н
General Data	Item	Number		Length	Tot Length	Sing Weight	Weight	Tot Weig
- Material Parameters and	No	lement	Section	[m]	[m]	[kg/m]	[kg]	[IJ
Design Parameters	ليسا	2	1 - IPE 550	6.000	12.000	105.190	631.14	1.
Results	2	2	3 - 15 650/190/9.4/14.6/21 / 4 - 15 45	2.414	4.829	84.953	205.10	U.
- Max Stresses by Section	3	2	2 - IPE 450	10.134	20.268	77.558	785.98	1
Max Stresses in Sets of	Sum	6			37.097			3
- Max Stresses in Element								
— Max Stresses in x Locati								
 Stresses in All Stress Po 								
 Governing Internal Force 								
crovening internal rolet								
Parts List								
Parts List	2.7.2 Par	rts List I	by Sets of Elements					_
Parts List	2.7.2 Par	rts List I B	by Sets of Elements	D	E	F	G	H
Parts List	2.7.2 Par A Item	rts List I B Number	by Sets of Elements C Set of Element	D Length	E Tot Length	F Sing Weight	G Weight	H Tot Wei
– Parts List	2.7.2 Par A Item No	rts List I B Number Sets	by Sets of Elements C Set of Element Description	D Length [m]	E Tot Length [m]	F Sing Weight [kg/m] 79.901	G Weight [kg]	H Tot Weij [t]
– Parts List	2.7.2 Par A Item No 1	rts List I B Number Sets 1	by Sets of Elements C Set of Element Description	D Length [m] 25.097	E Tot Length [m] 25.097	F Sing Weight [kg/m] 78.981	G Weight [kg] 1982.16	H Tot Wei [t] 1
Parts List	2.7.2 Par A Item No 1 Sum	rts List I B Number Sets 1 1	by Sets of Elements C Set of Element Description	D Length [m] 25.097	E Tot Length [m] 25.097 25.097	F Sing Weight [kg/m] 78.981	G Weight [kg] 1982.16	H Tot Weij [t] 1
Parts List	2.7.2 Pat A Item No 1 Sum	rts List I B Number Sets 1 1	oy Sets of Elements C Set of Element Description	D Length [m] 25.097	E Tot Length [m] 25.097 25.097	F Sing Weight [kg/m] 78.981	G Weight [kg] 1982.16	H Tot Wei [t] 1 1
Parts List	2.7.2 Pai A Item No 1 Sum	rts List I B Number Sets 1 1	by Sets of Elements C Set of Element Description	D Length [m] 25.097	E Tot Length [m] 25.097 25.097	F Sing Weight [kg/m] 78.981	G Weight [kg] 1982.16	H Tot Wei [t] 1
Parts List	2.7.2 Par A Item No 1 Sum	rts List I B Number Sets 1 1	oy Sets of Elements C Set of Element Description	D Length [m] 25.097	E Tot Length [m] 25.097 25.097	F Sing Weight [kg/m] 78.981	G Weight [kg] 1982.16	H Tot Wei [t] 1
Paris Liet	2.7.2 Par Item No 1 Sum	rts List I B Number Sets 1 1	oy Sets of Elements C Set of Element Description	D Length [m] 25.097	E Tot Length [m] 25.097 25.097	F Sing Weight [kg/m] 78.981	G Weight [kg] 1982.16	H Tot Weig [t] 1 1
Paris List	2.7.2 Par A Item No 1 Sum	rts List I B Number Sets 1 1	oy Sets of Elements C Set of Element Description	D Length [m] 25.097	E Tot Length [m] 25.097 25.097	F Sing Weight [kg/m] 78.981	G Weight [kg] 1982.16	H Tot Weig [t] 1 1
Paris List	2.7.2 Par A Item No 1 Sum	rts List I B Number Sets 1 1	oy Sets of Elements C Set of Element Description	D Length [m] 25.097	E Tot Length [m] 25.097 25.097	F Sing Weight [kg/m] 78.981	G Weight [kg] 1982.16	H Tot Weig [t] 1.
Paris List	2.7.2 Par A Item No 1 Sum	rts List I B Number Sets 1 1	oy Sets of Elements C Set of Element Description	D Length [m] 25.097	E Tot Length [m] 25.097 25.097	F Sing Weight [kg/m] 78.981	G Weight [kg] 1982.16	H Tot Weig [t] 1 1

Mask 2.7 Parts List

ASD automatically chooses an *Item Number* and displays in mask 2.7.1 the corresponding *Number of Elements*, Section, and *Length* of element and *Total Length* of all elements for each item number.

In mask 2.7.2, a parts list by sets of elements is listed. It is necessary for sets of elements to be previously defined in RSTAB, and, under *Details* in *ASD*, the appropriate option must also be activated. When using tapered sections or varying section types in a single set of elements the *Single Weight* is determined from the mean weight.

3.13. Pulldown Menus

The pulldown menus hold all the necessary functions to manage the design cases and results. A pulldown menu can be activated by just clicking on it or by using the shortcut by pressing the [Alt] key plus the underlined letter in the menu title. For example, to use the menu, File, just press [Alt+F] and the menu will roll down. This procedure applies to each pulldown menu.

File

...handles the design cases.

Rectance of the parameters and Design Parameters and Design Parameters and Design Parameters and Design Parameters and Marking Parameters and Design Parameters and Design Parameters and Design Parameters and Design Parameters and Parameter	L1 General Data Elements to Design: Example Load Systems No LS1 LS2 Schne LS4 Ingetted from exciption LS3 Selected for Description LS4 LS4 <t< th=""></t<>
Calculation Details	Comments

Pulldown Menu File

New [Ctrl+N]

... creates a new ASD Case.

New ASI)-Case			×
No	Description Stress Anal	lysis		•
	OK	Cancel	Help	

New ASD Case

To create a ASD case, you must enter a *Number* and a *Description*. [Arrow down] lists all previously used descriptions for your convenience. [OK] creates the new case.

Rename

...renames the current ASD case by changing its *Description* or *Number*.

Renam	e ASD-Case			X			
No 1	Description Stress Anal	Description					
	ОК	Cancel	Help				

Rename ASD Case

If you change the number of the ASD case, be sure to use a number currently not in use by another ASD case.

[Alt+F]

...deletes the case selected from a list of all existing ASD cases.



Delete Cases

<u>0</u>K

[Alt+E]

Select the case that should be deleted and press [OK].

Edit

... offers many functions related to the handling of sections.

ASD -	[Demo-4]										×
File	Edit Help										
ASE	Export Optimized Section	is to RSTA	48	's	1						
P	Pick New Section from Lit Edit List "Elements to De: Import Sections from R5" Section Properties	orary sign" TAB		•	B Yield Stress [kN/cm^2] 24.	Ten [kl 32	C sile Stress V/cm^2] 39.99	Modulus of [kN/c	Elasticity m^2] 19993.80	Co ASD, 9th editio	E ode n
8-1	Materials' Library										
	Go To Section Go To		Double Clic	<	_	_		-	_	Mater	ials' Library
	- Stresses in All Stress Po - Governing Internal Force - Parts List	Sec No 1 2 3 4	A Mat No 1 IPE 55 1 IPE 45 1 IS 650 1 IS 650	0 0 /190/9 /190/9	B Section Description 3.4/14.6/21 3.4/14.6/21		C Opti mize	Remark	IPE 550	7	
		Commen	ts			Sect	ion	Section Library	Displa	ay Stress Points	2,y
Calc	culation Details				« »		Graphic		ОК	Cancel	Help

Pulldown Menu Edit

Export Optimized Sections to RSTAB

...transfers the sections optimized in ASD to RSTAB. If the ASD calculation is started, all the necessary RSTAB results are calculated automatically as well. Please note that exporting sections from ASD to RSTAB also deletes all RSTAB results.

Import New Section from Library

...opens the section library. Here you can select a section for use in ASD. Note that it will not automatically be adopted in RSTAB at the same time!

Edit List "Elements to Design"

...offers a fast way to edit the list of *Elements to Design* in mask 1.1. The menu items under this function are only enabled if mask **1.2** *Materials and Sections* is open.

<u>Calculation</u>

ŀ	SD -	[Demo-4]										×
• 1	File	Edit Help										
1	ASE	Export Optimized Section	is to RST	AB		'5						
ł	-	Pick New Section from Lib	mary			В		0	D	61		E
d		Edit List "Elements to Des	ign"		•	Set only Elements	of this Secti	on in List	lus of kN/cr	Elasticity n^21	G	ode
		Import Sections from RS1 Section Properties	TAB			Add Elements of the	is Section to	o List on from Lir		19993.80	ASD, 9th editio	n
	-	Decidori Properdest.				Nelliove Lipitients	or one pecc	onnomus	·•			
		Materials' Library										
1		Go To Section			ible Click					_	Mate	nals' Library
1		Go To			r				-			
ł		- Stresses in All Stress Po	Sec	Mat		Section		Opti	D	IPE 550		
		Barts List	No	No		Description		mize	Remark			
:			2	1	IPE 550			<u> </u>				-
ł			3	1	IS 650/190.	/9.4/14.6/21					:	
1			4	1	IS 450/190.	/9.4/14.6/21						
1												2,7
1												
1												
			Comme	nts							3,5	
							Section	ູ່່່ິ	ection			
i I	•						Deta	ils	Library	🔲 Displa	y Stress Points	Numbering
	Ca	Iculation Details				<< >>>	Gra	aphic		ОК	Cancel	Help
S	iets o	inly elements with current sec	tion to lis	st "Eleme	ents to Desig	n".						

Pulldown Menu Edit→Edit List "Elements to Design"

Set only Elements of this Section in List

...sets only the elements of the current section to the list, "Elements to Design". The current section is determined by positioning the cursor. This function is useful if you only want to design a single section.

Add Elements of this Section to List

...adds the elements of the current section to the list, "Elements to Design". This function is useful for expanding an existing list with the elements of another section.

Remove Elements of this Section from List

... removes the elements of the current section from the list, "Elements to Design".

Import Section from RSTAB

...imports the section of the current Section Number from RSTAB. This function is useful to reset the ASD section with the one from RSTAB.

Section Properties

...displays all available properties of the current section.

Materials Library

... opens the window with the table of the entered allowable stresses.

Ma	Materials' Library								
Г		A	В	С	D	E	 		
	Mat No	Material Description	Yield Stress [kN/cm^2]	Tensile Stress [kN/cm^2]	Modulus of Elasticity [kN/cm^2]	Code			
	1	Steel A36	24.82	39.99	19993.80	ASD, 9th edition			
	2	Steel A529	28.96	41.37	19993.80	ASD, 9th Edition			
	3	Steel A441 G4,5	28.96	43.43	19993.80	ASD, 9th Edition			
	4	Steel A441 G3	31.71	46.19	19993.80	ASD, 9th Edition			
	5	Steel A441 G1,2	34.47	48.26	19993.80	ASD, 9th Edition			
	6	Steel A572 Grade 42	28.96	41.37	19993.80	ASD, 9th Edition	-		
	6 Steel A572 Grade 42 28.96 41.37 19993.80 ASD, 9th Edition ▼ New Edit Delete OK Cancel Help								

ASD Materials Library



Next to the *Material Number*, you see the *Material Description* and the corresponding *yield* and *tensile stresses*, moduli of elasticity and reference to the code of practise. You can select a material with all its parameters by placing the cursor into the corresponding line and pressing [OK] or [Enter]. With [New], a new material can also be entered into the library for use again later.

5et New Materia	al Parameters in Library	×
Material		_
Description: Ste	el A36 List No:	16
Code: AS	D, 9th edition	
Material Paramete	ers	
Yield Stress:	24.82 [kN/cm^2]	
Tension Stress:	39.99 [kN/cm^2]	
Modulus of Elasti	city: 19993.80 [kN/cm^2]	
	OK Cancel Help	

Set New Material in Library

<u>E</u>dit

In Description, enter the material name. The List Number is created automatically by ASD. Enter the Code and the respective material parameters. Confirm the input data with [OK].

In the Materials library, [Edit] opens a similar dialog.

Edit materia	l param	eters in lil	orary		×
Material —					
Description:	Steel A3	6		List No:	1
Code:	ASD, 9th	n edition			
Material Para	meters –				
Yield Stress:		24.82	[kN/cm	n^2]	
Tension Stre	ss:	39.99	[kN/cm	n^2]	
Modulus of E	lasticity:	19993.80	[kN/cm	n^2]	
			Cancel	Help	_
		<u>`</u>	Cancer		

Edit Material in Library

Here you can edit all previous described input including the List Number. The materials are sorted in the library by list number. Those numbers may be changed so that the materials most commonly used are placed at the top of the list.

With [Delete], you can delete the current or all selected materials from the library.

ASD	<u> </u>
⚠	Delete Material Parameter(s) from Library?
	Ja Nein

ASD Query

<u>Y</u>es

<u>D</u>elete

A deletion enquiry appears which must be confirmed. This helps prevent unwanted deletion occuring through a slip of the finger!

Go To Section

...jumps to the current section in mask 1.2.2.

Go To

...jumps to the desired location.



Go To				×
<u>G</u> o To				
Cross-see	etio 🚺	-		
No		Descrip	otion	
1	IPE 500			
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		<u> <u>u</u>k</u>	Lancel	Help

Go To

Depending on the mask, you can quickly jump to the desired *Element* or *Section*. Therefore, enter a number or select an item from the displayed list.

3.13.1. Help

[Alt+H]

... opens the help system.

3.14. Results

Graphic Display

OK and <u>G</u>raphic

After the calculation, the results can be displayed by pressing [Graphic]. The current ASD Case is shown.



Graphic Display for Results of ASD



After clicking on [Graphic], you'll immediately see the stress distribution and *ASD*,*Stress* window. In the stress window, it's possible to determine which stress is to be displayed and the relative size of the stress distribution contour with respect to the elements of the structure. With [Set] the changed settings are applied to the display. [All] applies the changed factor to all windows displaying stresses. [ASD] takes you back to the ASD masks.

If there is more than one case, the various ASD cases can be viewed by selecting them from the drop down list box in the main display. This is the same list box as the one that lets you flip through various load systems.



With [Print], you can print the graphic just like any other graphic of RSTAB, either to the printout report or directly to the printer.

To print the numerical results of ASD, a [Printout Report] must first be opened or created in RSTAB.



New Printout Report

Set up the printout report as described in RSTAB. It is created through the options *Printout Report, Printout Report Template,* and *Create Printout Report.* The report includes the ASD results and is processed by clicking on [OK].

The printout report is unique for RSTAB, ASD and all other additional modules. It is made so all data can be handled in one document. Since the data may be very large, it is important to work with selections and printout report templates. This speeds up the creation and display of the printout report and helps prevents your computer from running out of memory.



ASD Data and Results in the Printout Report

In ASD, you have all editing tools and options as described in the RSTAB manual. If you have a fully licensed version of ASD, you'll have more selection pages for the ASD data. The Selection can be accessed with *Edit* \rightarrow *Global Selection*. In the left hand window of the global selection window, select the *Program*, ASD.

<u>0</u>K

Printout Report	Selection D1				×
Program:	Global Selection Input Data Results				
RSTAB ASD STEEL EL-PL BGDK FE-BGDK FE-BEUL CONCRETE TIMBER DEFORM	Display ASD Data Imput D Result: ASD Cases to Display AI	ata S Display ASD Cases			
B-ZU-T	Existing ASD	Cases:	ASD Cases to Display:		
STIRNPL	No	Description	No	Description	
RAHMELK DSTV DOWEL DYNAM RSBUCK FUND			ASUT	Stress Analysis	
	Add>	Add All		<remove rem<="" th=""><th>ove Ali</th></remove>	ove Ali
Cover				OK Cancel	Help

ASD Selection – Main Selection

<u>A</u> dd>
Add All
< <u>R</u> emove
Re <u>m</u> ove All

 \bullet

In the tab folder, *Main Selection*, first select the main chapters for the printout report, found under *Display*. If you choose not to *Display All ASD Cases*, you can choose individual *Existing ASD Cases* with *ASD Cases to Display*. To shift single cases from one list to the other, use the buttons [Add], [Add All], [Remove] and [Remove All].

By checking the box, *Contents* you can include the ASD table of contents in the printout report.

Printout Report	Selection D1	×
Program:	Global Selection Input Data Results	
RSTAB ASD STEEL EL-PL BGDK FE-BEUL CONCRETE TIMBER DEFORM B-ZU-T STIRINPL RAHMECK DSTV DOWEL DYNAM RSBUCK FUND	Local Selection of ASD Cases: 1 Display General Data No Selection (e.g. 1-5,20) General Data No Selection (e.g. 1-5,20) Material Parameters Materials: All Stress Points Sections: All F Design Parameters Design: All V Design Parameters Design: All V Design Parameters Design: All V	
Contents	OK Cancel	Help



In the tab folder, *Input Data*, you can check the options *Display* of *General Data*, *Materials, Design parameters* and *Sections*, including their *Stress Points*. A more detailed selection is also possible. Just click on the [Arrow Down] button in the list boxes to the right of each topic and enter the numbers of the *Materials* and/or *Sections* in the empty line.

^o rogram:	Global Selection Input Data Results
Program: RSTAB ASD STEEL EL-PL BGDK FE-BGDK FE-BGDK FE-BGDK FE-BGDK CONCRETE TIMBER DEFORM B-ZU-T DEFORM B-ZU-T DSTV DOTV	Global Selection Input Data Results Local Selection of ASD Cases: 1 Display No Selection [e g 15,20] If Max Stresses by Section Sections: AI If Max Stresses by Set of Elements: AI If Max Stresses by Set of Elements: AI If Max Stresses by Location Elements: If Max Stresses by Location Elements: If Max Stresses by Location Elements: If Governing Internal Forces Elements: If Parts List by Elements Items: If Occes Elements: If Parts List by Elements Items: If Dosign Detail in Elements Elements: If Design Detail in Elements Elements:

ASD Selection – Results

Finally, in the tab folder, *Results*, make a selection for results under *Display*. Again, displaying all or just single items can be chosen and applied.

In every tab folder, [OK] confirms the settings and closes the window. The printout report is then rebuilt. [Cancel] closes the dialog without adopting the changes.

Appendix A: ASD Design Equations

A.1 Chapter D - Tension Members

A1.1 D1. Allowable stresses

 $Ft \leq 0.6 Fy$ (Gross Area - Agross)

 $Ft \leq 0.5 Fu$ (Net Area - Anet)

In addition, pin-connected members shall meet the requirements of Sect. D3.1 at the pin hole

A1.2 D3.1 Pin-connected members

The allowable stress on the net area of the pin hole for pin-connected members is



 $Ft \leq \ 0.45 \ Fy$

The allowable stresses on the eyebars meeting the requirements of Sect. D3.3 is ;

 $Ft \le 0.60 Fy$ (on the body area)

A.2 Chapter E - Columns and other compression members

A2.1 E1. Effective Length and Slenderness Ratio

The effective-length factor K shall be determined in accordance with Sect. C2

Kl =	effective length	where l is the actual length of the member
r =	radius of gyration	

Kl/r = slenderness ratio. For limiting slenderness ratios, see Sect. B7

A2.2 E2. Allowable Stresses

For axially loaded compression members whose cross-sections meet the provisions of Table B5.1, when the largest slenderness ratio Kl/ r $\,<$ C_c, the allowable stress is ;

$$F_{a} = \frac{\left[1 - \frac{(Kl/r)^{2}}{2C_{c}^{2}}\right]F_{y}}{\frac{5}{3} + \frac{3(Kl/r)}{8C_{c}} - \frac{(Kl/r)^{3}}{8C_{c}^{3}}}$$
(E2-1)

where
$$C_c = \sqrt{\frac{2\pi^2 E}{F_y}}$$

When Kl/r exceeds C_c , the allowable stress is;

$$F_a = \frac{12\pi^2 E}{23(Kl/r)^2}$$
(E2-2)



A.3 Chapter F – Beams and other flexural members

A3.1 F1. Allowable Stress: Strong Axis Bending of I-Shaped Members and Channels

1. Members with Compact Sections

For members with compact sections (excluding hybrid members and members with yield points greater than 65 ksi), symmetrical about, and loaded in the plane of their minor axis the allowable stress is

$$F_b = 0.66 F_y$$
 (F1-1)

provided the flanges are connected continuously to the web or webs and the laterally unsupported length of the compression flange L_b does not exceed the value of L_c , as given by the smaller of

$$\frac{76b_f}{\sqrt{F_y}} \quad or \quad \frac{20\,000}{(d/A_f)F_y} \tag{F1-2}$$

2. Members with Noncompact Sections

For members meeting the requirements of Sect. F1.1 except that their flanges are noncompact (excluding built-up members and members with yield points greater than 65 ksi), the allowable stress is

$$F_b = F_y \left[0.79 - 0.002 \frac{b_f}{2t_f} \sqrt{F_y} \right]$$
 (F1-3)

For built-up members meeting the requirements of Sect. F1.1 except that their flanges are noncompact and their webs are compact or noncompact, (excluding hybrid girders and members with yield points greater than 65 ksi) the allowable stress is

$$F_{b} = F_{y} \left[0.79 - 0.002 \frac{b_{f}}{2t_{f}} \sqrt{\frac{F_{y}}{k_{c}}} \right]$$
(F1-4)

where
$$k_c = \frac{4.05}{(h/t_w)^{0.46}}$$
 if $h/t_w > 70$, otherwise $k_c = 1.0$

For members with noncompact section (Sect. B5), but not included above, and loaded through the shear center and braced laterally in the region of compression stress at intervals not exceeding 76 $b_f / \sqrt{F_v}$

$$F_b = 0.6 F_v$$

Members with Compact or Noncompact Sections with Unbraced Length Greater than $L_{\rm c}$

For flexural members with compact or noncompact sections as defined in Sect B5.1, and with unbraced lengths greater than L_c as defined in Sect. F1.1, the allowable bending stress in tension is determined from Equation (F1-5).

For such members with an axis of symmetry in, and loaded in the plane of their web, the allowable bending stress in compression is determined as the larger value from Equations (F1-6) or (F1-7) and (F1-8), except that Equation (F1-8) is only applicable to sections with a compressive flange that is solid and approximately rectangular in cross-section and that has an area not less than the tension flange. Higher values of the allowable compressive stress are permitted if justified by a more precise analysis. Stresses shall not exceed those permitted by Chapter G, if applicable.

For channels bent about their major axis, the allowable compressive stress is determined from Equation (F1-8).

When

$$\sqrt{\frac{102 \times 10^{3} C_{b}}{F_{y}}} \le \frac{l}{r_{T}} \le \sqrt{\frac{510 \times 10^{3} C_{b}}{F_{y}}}$$

$$F_{b} = \left[\frac{2}{3} - \frac{F_{y} \left(l/r_{T}\right)^{2}}{1530 \times 10^{3} C_{b}}\right] F_{y} \leq 0.60 F_{y}$$
(F1-6)

When

$$\frac{l}{r_T} \ge \sqrt{\frac{510 \times 10^3 C_b}{F_y}}$$

$$F_{b} = \frac{170 \times 10^{3} C_{b}}{\left(l/r_{T}\right)^{2}} F_{y} \leq 0.60 F_{y}$$
(F1-7)

For any value of l/r_T :

$$F_b = \frac{12 \times 10^3 C_b}{ld / A_f} \le 0.60 F_y$$
(F1-8)

where

l = distance between cross-sections braced against twist or lateral displacement of the compression flange, in. For cantilevers braced against twist only at the support, l may be conservatively be taken as the actual length.

 r_T = radius of gyration of a section comprising the compression flange plus 1/3 of the compression web area, taken about an axis in the plane of the web, in.

 A_f = area of the compression flange, in².

 $C_b = 1.75 + 1.05 (M_1/M_2) + 0.3 (M_1/M_2)^2$, but not more than 2.3, where M_1 is the smaller and M_2 the larger bending moment at the ends of the unbraced length, taken about the strong axis of the member, and where M_1/M_2 , the ratio of end moments, is positive when M_1 and M_2 have the same sign (single curvature bending) and negative when they are of opposite signs (reverse curvature bending). When the bending moment at any point within the unbraced length is larger than that at boh ends of this length, the value of C_b shall be taken as unity.

C_b may be taken conservatively as unity.

For hybrid plate girders, F_y for Equations (F1-6) and (F1-7) is the yield stress of the compression flange.

Equation (F1-8) shall not apply to hybrid girders.

Sect. F1.3 does not apply to tee sections if the stem is in compression anywhere along the unbraced length.

A3.2 F2. Allowable Stress: Weak Axis Bending of I-Shaped Members, Solid Bars and Rectangular Plates

Lateral bracing is not required for members loaded through the shear center about their weak axis nor for members of equal strength about both axes.

Members with Compact Sections

For doubly symmetrical I- and H-shape members with compact flanges (Sect. B5) continuously connected to the web and bent about their weak axis (except members with yield points greater than 65 ksi); solid round and square bars; and solid rectangular sections bent about their weaker axis, the allowable stress is

$$F_b = 0.75 F_y \tag{F2-1}$$

Members with Noncompact Sections

For members not meeting the requirements for compact sections of Sect. B5 and not covered in Sect. F3, bent about their minor axis, the allowable stress is

$$F_b = 0.60 F_y$$
 (F2-2)

Doubly symmetrical I- and H-shapes members bent about their weak axes (except members with yield points greater than 65 ksi) with noncompact flanges (Sect. B5) continuously connected to the web may be designed on the basis of an allowable stress of

$$F_{b} = F_{y} \left[1.075 - 0.05 \left(\frac{b_{f}}{2t_{f}} \right) \sqrt{F_{y}} \right]$$
 (F2-3)

A3.3 F3.Allowable Stress: Bending of Box Members, Rectangular Tubes and Circular Tubes

Members with Compact Sections

For members bent about their strong or weak axes, members with compact sectionsas defined in Sect. B5 and flanges continuously connected to the webs, the allowable stress is

$$F_b = 0.66 F_y$$
 (F3-1)

To be classified as a compact section, a box-shaped member shall have, in addition to the requirements in Sect. B5, a depth not greater than 6 times the width, a flange

thickness not greater than 2 times the web thickness and a laterally unsupported length L_{b} less than or equal to

$$L_{c} = \left(1950 + 1200 \frac{M_{1}}{M_{2}}\right) \frac{b}{F_{y}}$$
(F3-2)

except that it need not be less than 1200 (b / F_y), where M_1 is the smaller and M_2 the larger bending moment at the ends of the unbraced length, taken about the strong end of the member, and where M_1/M_2 , the ratio of the end moments, is positive when M_1 and M_2 have the same sign (single curvature bending) and negative when they are of opposite signs (reverse curvature bending).

Members with Noncompact Sections

For box-type and tubular flexural members that meet the noncompact section requirements of Sect. B5, the allowable stress is

$$F_b = 0.60 F_y$$
 (F3-3)

Lateral bracing is not required for a box-section whose depth is less than 6 times ist width. Lateral-support requirements for box sections of larger depths-to-width ratios must be determined by special analysis.

A3.4 F4. Allowable Shear Stress

For h/tw $\leq 380 / \sqrt{F_y}$, on the overall depth times the web thickness, the allowable shear stress is

$$F_v = 0.40 F_v$$
 (F4-1)

For h/tw > 380 / $\sqrt{F_y}$, the allowable shear stress is on the clear distance between flanges time the web thickness is

$$F_{\nu} = \frac{F_{\nu}}{2.89} (C_{\nu}) \le 0.4 F_{\nu}$$
 (F4-2)

where

$$C_{v} = \frac{45000 k_{v}}{F_{y} (h/t_{w})^{2}} \quad \text{when } C_{v} \text{ is less than } 0.8$$
$$= \frac{190}{h/t_{w}} \sqrt{\frac{k_{v}}{F_{y}}} \quad \text{when } Cv \text{ is greater than } 0.8$$



- tw = thickness of web, in
- a = clear distance between transverse stiffeners, in.
- h = clear distance between flanges at the section under investigation, in.

Maximum h/tw limits are given in Chapter G.

An alternative design method for plate girders utilizing tension field action is given in Chapter G.

A.4 Chapter G – Plate Girders

A4.1 G2. Allowable Stresses

When the web depth-to-thickness ratio exceeds 970 / $\sqrt{F_y}$, the maximum bending stress in the compression flange shall not exceed

$$F_b = F_b R_{PG} R_e \tag{G2-1}$$

where

 F_b = applicable bending stress given in Chapter F, ksi

$$R_{PG} = 1 - 0.005 \frac{Aw}{Af} \left(\frac{h}{t} - \frac{760}{\sqrt{F_b}} \right) \le 1.0$$

$$R_e = \frac{12 + \left(\frac{A_w}{A_f}\right) (3\alpha - \alpha^3)}{12 + 2\left(\frac{A_w}{A_f}\right)} \le 1.0$$

(non-hybrid girders, $R_e = 1.0$)

 A_w = area of web at he section under investigation, in.²

 $/ \searrow$

 $A_f =$ area of compression flange, in.²

$$\alpha = 0.6 F_{vb}/F_b \le 1.0$$

A4.2 G3. Allowable Shear Stress with Tension Field Action

Except as herein provided, the largest average web shear, f_v , in kips per sq. In., computed for any condition of complete of partial loading, shall not exceed the value given by Equation (F4-2)

Alternatively, for girders other than hybrid girders, if intermediate stiffeners are provided and spaced to satisfy the provisions of Sect. G4 and if $C_v \leq 1$, the allowable shear including tension field action given by Equation (G3-1) is pemitted in lieu of the value given by Equation (F4-2).

$$F_{\nu} = \frac{F_{\nu}}{2.89} \left[C_{\nu} + \frac{1 - C_{\nu}}{1.15\sqrt{1 + (a/h)^2}} \right] \le 0.40 F_{\nu}$$
(G3-1)*

* Equation (G3-1) recognises the contribution of tension field action.

A4.3 G5. Combined Shear and Tension Stress

Plate girder webs which depend upon tension field action, as provided in Equation (G3-1), shall be so proportioned that bending tensile stress, due to moment in the plane of the girder web, shall not exceed $0.60F_v$ nor

$$\left(0.825 - 0.375 \frac{f_v}{F_v}\right) F_y$$
(G5-1)

where

 $f_v =$ computed average web shear stress (total shear divided by web area)

 F_v = allowable web shear stress according to Equation (G3-1), ksi

The allowable shear stress in the webs of girders having flanges and webs with yield point greater than 65 ksi shall not exceed the values given by Equation (F4-2) if the flexural stress in the flange f_b exceeds 0.75 F_b .

A.5 Chapter H – Combined Stresses

This chapter pertains to doubly and singly symmetrical sections only. See Chapter E for determination of F_a and Chapter F for determination of F_{bx} and F_{by} .

A5.1 H1 Axial Compression and Bending

Members subjected to both axial compression and bending stresses shall be proportioned to satisfy the following requirements:

$$\frac{f_a}{F_a} + \frac{C_{mx} f_{bx}}{\left(1 - \frac{f_a}{F'_{ex}}\right) F_{bx}} + \frac{C_{my} f_{by}}{\left(1 - \frac{f_a}{F'_{ey}}\right) F_{by}} \le 1.0$$
(H1-1)

$$\frac{f_a}{0.6F_y} + \frac{f_{bx}}{F_{bx}} + \frac{f_{by}}{F_{by}} \le 1.0$$
(H1-2)

When fa/Fa ≤ 0.15 , Equation (H1-3) is permitted inlieu of Equations (H1-1) and (H1-2) :

$$\frac{f_a}{F_a} + \frac{f_{bx}}{F_{bx}} + \frac{f_{by}}{F_{by}} \le 1.0$$
(H1-3)

In equations (H1-1), (H1-2) and (H1-3), the subscripts x and y, combined with subscripts b, m and e, indicate the axis of bending about which a particular stress or design property applies, and

 F_a = axial compressive stress that would be permitted if axial force alone existed, ksi.

 F_b = compressive bending stress that would be permitted if bending moment alone existed, ksi

F'_e =
$$\frac{12\pi^2 E}{23(Kl_h / r_h)^2}$$

= Euler stress divided by a factor of safety, ksi (In the expression for F'e,lb is the actual unbraced length in the plane of bending and rb is the corresponding radius of gyration. K is the effective length factor in the plane of bending.) As in the case of Fa, Fb and 0.6Fy, F'e may be increases 1/3 in acordance with Sect. A5.2



 $f_a =$ computed axial stress, ksi

 f_b = computed compressive bending stress at the point under consideration, ksi

 C_m = Coefficient whose value shall be taken as follows:

For compression members in frames subject to joint translation (sidesway), $C_m = 0.85$

For rotationally restrained compression members in frames braced against joint translation and not subject to transverse loading between their supports in the plane of bending,

 $C_m = 0.6 - 0.4 (M_1/M_2)$

where M_1/M_2 is the ratio of the smaller to larger moments at the ends of that portion of the member unbraced in the plane of bending under consideration. M_1/M_2 is positive when the member is bent in reverse curvature, negative when bent in single curvature.

For compression members in frames braced against joint translation in the plane of loading and subjected to transverse loading between their supports, the value of Cm may be determined by an analysis. However, in lieu of such an analysis, the folloowing values are permitted:

i. For members whose ends are restrained against rotation in the plane of bending : $C_m = 0.85$

ii. For members whose ends are unrestrained against rotaion in the plane of bending: $C_m = 1.0$

A5.2 H2. Axial Tension and Bending

Members subjected to both axial tension and bending stressess shall be proportioned at all points along their length to satisfy the following equation:

$$\frac{f_a}{F_t} + \frac{f_{bx}}{F_{bx}} + \frac{f_{by}}{F_{by}} \le 1.0$$
 (H2-1)

A.6 Appendix B – Design Requirements

A6.1 B5. Local Buckling

Stiffened Compression Elements

For axially loaded circular sections :

Members with diameter-to-thickness ratios D/t greater than 3,300/Fy, but having a diameter-to-thickness ratio of less than 13,000/Fy, shall not exceed the smaller value determined by Sect. E2 nor

$$F_a = \frac{662}{D/t} + 0.40 F_y$$
 (A-B5-9)

A.7 Appendix F – Beams and other Flexural Members

A7.1 F7. Web-Tapered Members

Allowable Compressive Stress

On the gross section of axially loaded tapered compression members, the allowable compressive stress, in kips per sq. In., shall not exceed the following:

When the effective slenderness ratio S is less than C_c :

$$F_{a\gamma} = \frac{\left(1.0 - \frac{S^2}{2C_c^2}\right)F_{\gamma}}{\frac{5}{3} + \frac{3S}{8C_c} - \frac{S^3}{8C_c^3}}$$
(A-F7-2)

When the effective slenderness ratio S exceeds Cc :

$$F_{a\gamma} = \frac{12 \pi^2 E}{23 S^2}$$
(A-F7-3)

Allowable Flexural Stress**

Tension and compression stresses on extreme fibres of tapered flexural members, in kips per sq. in. , shall not exceed the following values:

$$F_{b\gamma} = \frac{2}{3} \left[1.0 - \frac{F_y}{6B\sqrt{F_{s\gamma}^2 + F_{w\gamma}^2}} \right] F_y \le 0.60 F_y$$
(A-F7-

unless $F_{b\gamma} \leq F_y / 3$, in which case

$$F_{b\gamma} = B\sqrt{F_{s\gamma}^{2} + F_{w\gamma}^{2}}$$
 (A-F7-5)

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In the above equations,

$$F_{s\gamma} = \frac{12 \times 10^3}{h_s L d_o / A_f}$$
(A-F7-6)

$$F_{w\gamma} = \frac{170 \times 10^3}{h_w L / r_{To}}$$
(A-F7-7)

where

hs = factor equal to $1.0 + 0.0230\gamma \sqrt{Ld_o / A_f}$ hw = factor equal to $1.0 + 0.0385\gamma \sqrt{L / r_{To}}$

 r_{To} = radius of gyration of a section at the smaller end, considering only the compression flange plus 1/3 of the compression web area, taken about an axis in the plane of the web, in.

 A_f = area of the compression flange, in.²

And where B is determined as follows

When the maximum moment M2 in three adjacent segments of approximately equal unbraced length is located within the central segment and M1 is the larger moment at one end of the three-segment portion of a member:*

$$B = 1.0 + 0.37 \left(1.0 + \frac{M_1}{M_2} \right) + 0.50\gamma \left(1.0 + \frac{M_1}{M_2} \right) \ge 1.0$$
 (A-F7-8)

When the largest computed bending stress f_{b2} occurs at the larger end of two adjacent segments of approximately equal unbraced lengths and f_{b1} is the computed bending stress at the smaller end of the two-segment portion of the member:*

$$B = 1.0 + 0.58 \left(1.0 + \frac{f_{b1}}{f_{b2}} \right) + 0.70\gamma \left(1.0 + \frac{f_{b1}}{f_{b2}} \right) \ge 1.0$$
 (A-F7-9)

When the largest computed bending stress f_{b2} occurs at the smaller end of two adjacent segments of approximately equal unbraced lengths and f_{b1} is the computed bending stress at the larger end of the two-segment portion of the member:**

$$B = 1.0 + 0.55 \left(1.0 + \frac{f_{b1}}{f_{b2}} \right) + 2.20\gamma \left(1.0 + \frac{f_{b1}}{f_{b2}} \right) \ge 1.0$$
 (A-F7-10)

In the foregoing, $\gamma = (d_L - d_o)/d_o$ is calculated for the unbraced length containing the maximum computed bending stress.

When the computed bending stress at the smaller end of a tapered member or segment thereof is equal to zero:

$$B = \frac{1.75}{1.0 + 0.25\sqrt{\gamma}} \tag{A-F7-11}$$

where $\gamma = (d_L - d_o)/d_o$, calculated for the unbraced length adjacent to the point of zero bending stress.

 M_1/M_2 is considered as negative when producing single curvature. In the rare cases where M_1/M_2 is positive, it is recommended it be taken as zero.

** f_{b1}/f_{b2} is considered as negative when producing single curvature. If a point of contraflexure occurs in one of two adjacent unbraced segments, f_{b1}/f_{b2} is considered as positive. The ratio $f_{b1}/f_{b2} \neq 0$.

Allowable Shear

The allowable shear stress of tapered flexural members shall be in accordance with Sect. F4.

Combined Flexure and Axial Force

Tapered members and unbraced segments thereof subjected to both axial compression and bending stresses shall be proportioned to satisfy the following requirement:

$$\left(\frac{f_{ao}}{F_{a\gamma}}\right) + \frac{C'_{m}}{\left(1 - \frac{f_{ao}}{F'_{e\gamma}}\right)} \left(\frac{f_{bl}}{F_{b\gamma}}\right) \leq 1.0$$
(A-F7-12)

and

$$\frac{f_a}{0.60 F_y} + \frac{f_b}{F_{b\gamma}} \le 1.0$$
 (A-F7-13)

When $f_{ao}/F_{a\gamma}$ \leq 0.15, Equation (A-F7-14) is permitted in lieu of Equations (A-F7-12) and (A-F7-13)

$$\left(\frac{f_{ao}}{F_{a\gamma}}\right) + \left(\frac{f_{bl}}{F_{b\gamma}}\right) \le 1.0 \tag{A-F7-}$$
14)

where

 $Fa\gamma$ = axial compressive stress permitted in the absence of bending moments, ksi

 $Fa\gamma$ = bending stress permitted in the absence of axial force, ksi

 $F'e\gamma$ = Euler stress divided by factor of safety, ksi, equal to

$$\frac{12\pi^2 E}{23(K_{\gamma}l_b / r_{bo})^2}$$

where l_b is the actual unbraced length in the plane of bending and r_{bo} is the corresponding radius of gyration at ist smaller end.

 f_{ao} =computed axial stress at the smaller end of the member or unbraced segment thereof, as applicable, ksi

 C_{m} ' = coefficient applied to bending term in interaction equation

$$= 1.0 + 0.1 \left(\frac{f_{ao}}{F'_{e_{\gamma}}} \right) + 0.3 \left(\frac{f_{ao}}{F'_{e_{\gamma}}} \right)^2$$

when the member is subjected to end moments which cause single curvature bending and approximately equal computed bending stresses at the ends

$$= 1.0 + 0.9 \left(\frac{f_{ao}}{F'_{e_{\gamma}}}\right) + 0.6 \left(\frac{f_{ao}}{F'_{e_{\gamma}}}\right)^2$$

when the computed bending stress at the smaller end of the unbraced length is equal to zero.

When $Kl/r \ge C_c$ and combined stresses are checked incrementally along the length, f_{ao} may be replaced by f_a , and f_{bl} may be replaced by f_b , in Equations (A-F7-12) and (A-F7-14)